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並不得書寫、畫記、作答。

國立清華大學 108 學年度碩士班考試入學試題

系所班組別：動力機械工程學系 乙組

考試科目(代碼)：控制系統(1202)

— 作答注意事項 —

1. 請核對答案卷(卡)上之准考證號、科目名稱是否正確。
2. 作答中如有發現試題印刷不清，得舉手請監試人員處理，但不得要求解釋題意。
3. 考生限在答案卷上標記「由此開始作答」區內作答，且不可書寫姓名、准考證號或與作答無關之其他文字或符號。
4. 答案卷用盡不得要求加頁。
5. 答案卷可用任何書寫工具作答，惟為方便閱卷辨識，請儘量使用藍色或黑色書寫；答案卡限用 2B 鉛筆畫記；如畫記不清(含未依範例畫記)致光學閱讀機無法辨識答案者，其後果一律由考生自行負責。
6. 其他應考規則、違規處理及扣分方式，請自行詳閱准考證明上「國立清華大學試場規則及違規處理辦法」，無法因本試題封面作答注意事項中未列明而稱未知悉。

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- Q1.** Figure 1 shows a rotor supported by two magnetic bearings. Bearing 1 is located at distance ℓ_1 from the center of mass of the rotor and generates a force f_1 to support the rotor, and bearing 2 is located at distance ℓ_2 from the center of mass of the rotor and generates a force f_2 to support the rotor. Denote the displacement of the center of mass of the rotor as x_c and the rotation angle of the rotor as θ . The mass of the rotor is m , and the moment of the inertia is I . Ignore the gravity and assume that the rotation angle θ is small so that the moment arm from bearing force f_1 to the center of mass is ℓ_1 , and the moment arm from bearing force f_2 to the center of mass is ℓ_2 .

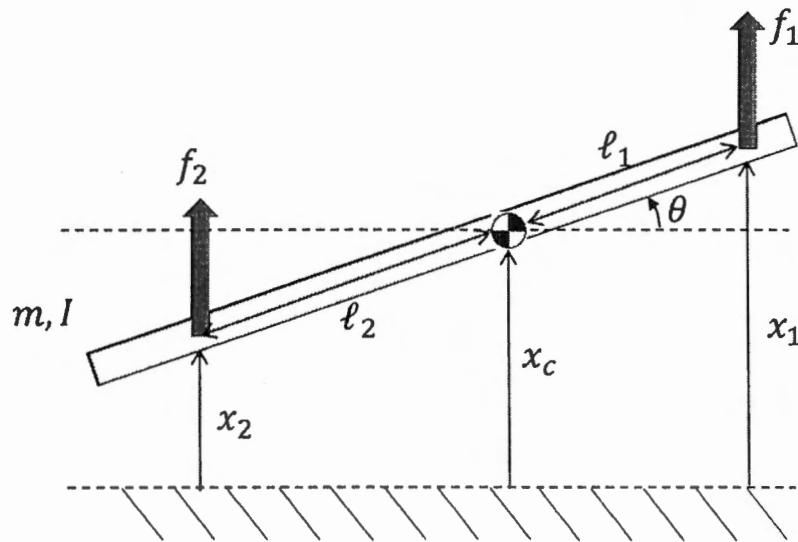


Figure 1

- (a) Using Newton's law, derive the differential equations to describe the dynamics of x_c and θ . **(2 pts)** Let $\mathcal{L}\{x_c(t)\} = X_c(s)$, $\mathcal{L}\{\theta(t)\} = \Theta(s)$, $\mathcal{L}\{f_1(t)\} = F_1(s)$, and $\mathcal{L}\{f_2(t)\} = F_2(s)$. From the differential equations you derived, it can be shown that

$$\begin{bmatrix} X_c(s) \\ \Theta(s) \end{bmatrix} = \begin{bmatrix} G_{11}(s) & G_{12}(s) \\ G_{21}(s) & G_{22}(s) \end{bmatrix} \begin{bmatrix} F_1(s) \\ F_2(s) \end{bmatrix}.$$

Compute $G_{11}(s)$, $G_{12}(s)$, $G_{21}(s)$, and $G_{22}(s)$. **(4 pts)**

- (b) Because of the electro-magnetic interaction, the magnetic forces can be expressed as $f_1 = k_x x_1 + k_i i_1$, and $f_2 = k_x x_2 + k_i i_2$, where $k_x > 0$ and $k_i > 0$ are electromagnetic coefficients, $x_1(x_2)$ is the rotor's displacement at

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the bearing 1 (2) (See Figure 1), and $i_1(i_2)$ is the control input current to bearing 1 (2). Assume θ is small, derive x_1 as a linear combination of x_c and θ , and x_2 as a linear combination of x_c and θ . **(2 pts)** Under the assumption that $I = m\ell_1\ell_2$, Compute $P_{11}(s)$, $P_{12}(s)$, $P_{21}(s)$, and $P_{22}(s)$ in the system

$$\begin{bmatrix} X_1(s) \\ X_2(s) \end{bmatrix} = \begin{bmatrix} P_{11}(s) & P_{12}(s) \\ P_{21}(s) & P_{22}(s) \end{bmatrix} \begin{bmatrix} I_1(s) \\ I_2(s) \end{bmatrix}. \quad \textbf{(12 pts)}$$

- (c) Using the transfer functions you find in part (b), compute the open-loop poles of the system. Is the system stable? **(5 pts)**

Q2. Consider a control system in Figure 2 in which $P(s) = \frac{1}{s^2-1}$.

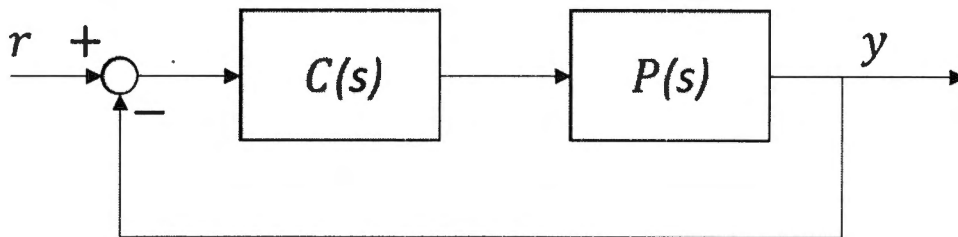


Figure 2

- (a) Plot the Bode plot of $P(s)$. **(4 pts)**
- (b) The target is to stabilize $P(s)$ using a lead compensator $C(s) = K \frac{(1+aTs)}{(1+Ts)}$ in which $K, a, T > 0$. Assuming that the steady-state error due to a step reference input should be limited to be within 4%, what is the minimum value for K ? **(3 pts)** Under the choice of the minimum K , it is desired that the phase margin is 45° . Determine the parameters a and T . What is the crossover frequency you achieve? You may want to conduct the compensator design using the Bode plot in (a). **(13 pts)**
- (c) Is it possible to achieve the same design specifications using a lag compensator? Show your reasoning. **(5 pts)**

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Q3.

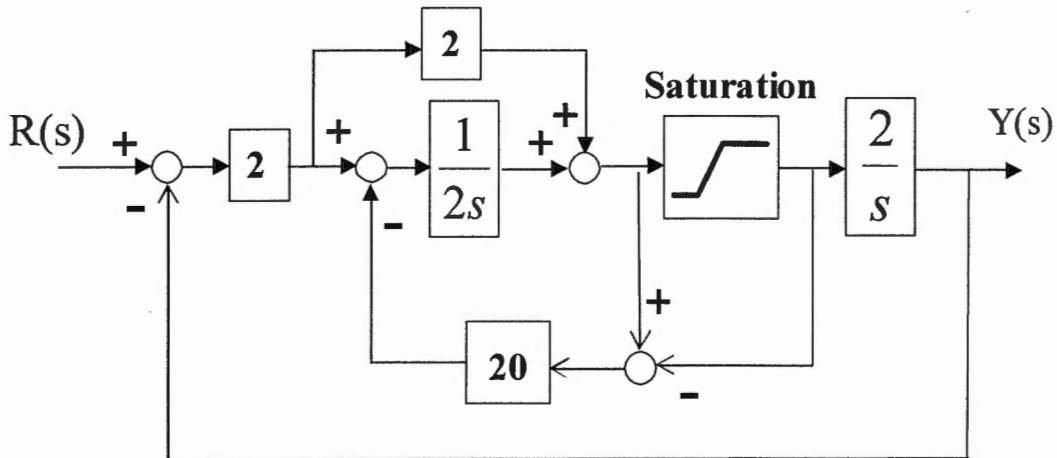


Figure 3

The block diagram above shows an example of anti-windup for integral control system. If we consider the saturation element as an equivalent gain, the root locus can be drawn and a prediction of the response to large inputs can be made.

- Compute the roots of the closed loop characteristic equation if the saturation block has a gain of 1 **(5 pts)**
- Replace the saturation block with an equivalent gain K (the slope is K before saturation). Sketch the root locus with respect to K ($K > 0$). **(10 pts)**
- For (b), calculate and give the break-away points and break-in points if there is any. **(5 pts)**
- Based on looking at the root locus, what is the largest value of slope K at the saturation block to result in a stable closed-loop system? **(5 pts)**

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Q4.

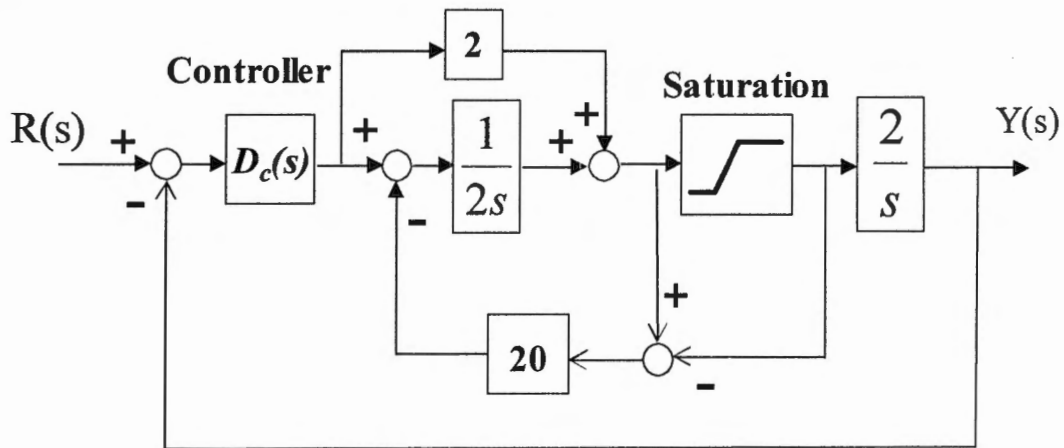


Figure 4

Ziegler-Nichols Tuning for the Regulator

$D_c(s) = k_p(1 + 1/T_I s + T_D s)$, Based on the Ultimate Sensitivity Method

| Type of Controller | Optimum Gain |
|--------------------|--|
| P | $k_p = 0.5K_u$ |
| PI | $\begin{cases} k_p = 0.45K_u \\ T_I = \frac{P_u}{1.2} \end{cases}$ |
| PID | $\begin{cases} k_p = 0.6K_u \\ T_I = 0.5P_u \\ T_D = 0.125P_u \end{cases}$ |

Table 1 Ziegler-Nichols Tuning method

- (a) In the case, the slope of the saturation element is 2 and the saturation signal is too large to be reached. Based on the Ziegler-Nichols Tuning method shown as above table, design a PI controller for $D_c(s)$ shown in above block diagram. Give the values of your (k_p, T_I) (10 pts)
- (b) Determine the **relevant error constant** with respect to reference input $R(s)$ for this closed-loop system with your PI controller $D_c(s)$ (5 pts)

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Q5. Given the state space representation of the plant.

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \end{bmatrix} = \begin{bmatrix} -1 & 1 \\ 2 & -2 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \end{bmatrix} + \begin{bmatrix} 0 \\ 1 \end{bmatrix} u$$

$$y = \begin{bmatrix} 50 & 0 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \end{bmatrix}$$

Design a state feedback controller such that the closed-loop response to a unit-step input has

$$M_p = 4.33\%, \quad t_s = 0.0474 \text{ sec}$$

(10 pts)