

1. A rotational system is shown in Fig. 1. Derive the differential equation for the motion in which  $\theta_o$  is the output and  $\theta_i$  is the input. Note that  $c$ ,  $k$ , and  $I$  are the damping coefficient, spring constant, and inertia of the mass, respectively. Also notice that  $n_1$  and  $n_2$  are the tooth number of gears whose inertia are negligible. (10%)

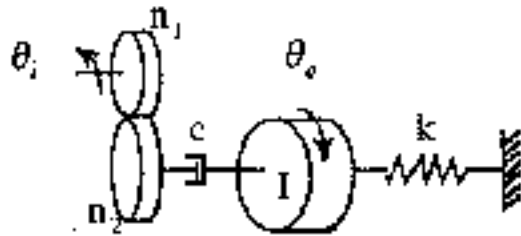


Fig. 1

2. A feedback control system is shown in Fig. 2 (a). The open-loop transfer function is

$$G(s) = \frac{(s+1)}{s(s+3)(s+5)}$$

- (a) With  $K = 150$  and the sensor gain  $\alpha = 1$ , determine the steady-state error of the closed-loop system for the periodic input shown in Fig. 2 (b). (7%)  
 (b) Find the sensitivity of the closed-loop system with the sensor gain  $\alpha$  and discuss the effect of  $\alpha$  to the system. (8%)

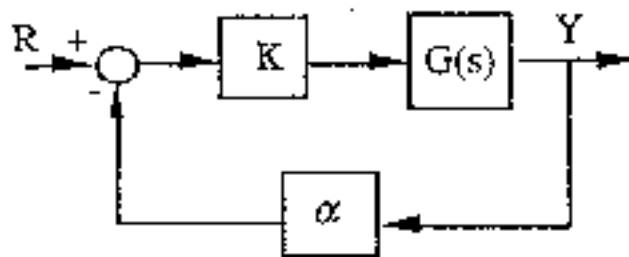


Fig. 2 (a)

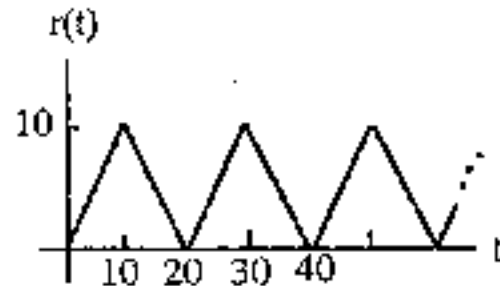


Fig. 2 (b)

3. The Nyquist plot for the open-loop system  $KG(s)$  is given in Fig. 3 where gain  $K = 2$ . Based on this plot, determine the range of values of  $K$  for which the system will be stable using Nyquist Stability criterion. (10%)

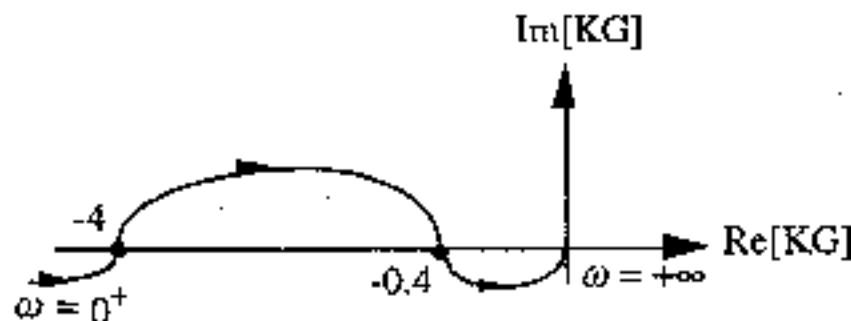


Fig. 3

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4. A dc motor shown in Fig. 4(a) is tested in open loop. When unloaded, its velocity response to a step voltage of 5 V is

$$\omega(t) = 100(1 - e^{-10t}) \text{ rad/s}$$

When the motor is loaded by  $T = 6 \text{ N-m}$ , its steady-state velocity drops to 70 rad/s to a step voltage of 5 V. The torque-velocity characteristic curve (assuming a straight line) of 5 V is shown in Fig. 4 (b).

- (a) Derive the equation of motion for the motor. Note that the disturbance (load) MUST be included. (7%)  
 (b) The motor is controlled using a PI controller as shown in Fig. 4(c). The tachometer gain is 0.5 V/rad. Determine the proportional gain  $k_p$  and integral gain  $k_i$  such that the system has damping ratio  $\zeta = 0.707$  and natural frequency  $\omega_n = 10 \text{ rad/s}$ . (8%)

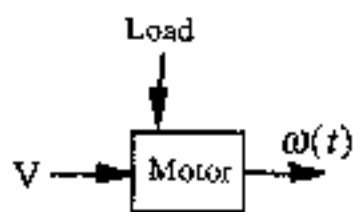


Fig. 4 (a)

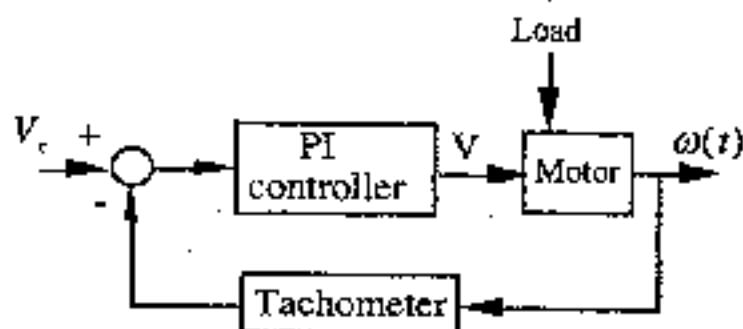


Fig. 4 (c)

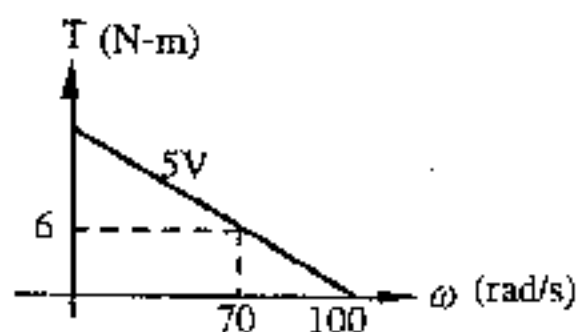


Fig. 4 (b)

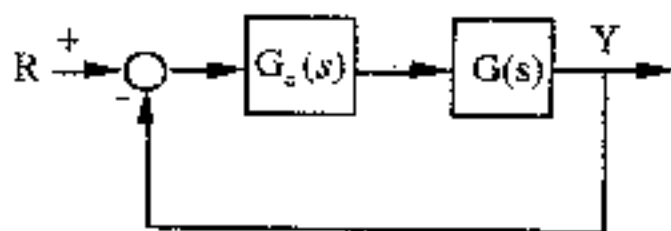


Fig. 5

5. A compensator is given as

$$D(s) = \frac{Ts + 1}{\alpha Ts + 1}, \quad 0 < \alpha < 1, \quad T > 0.$$

- (a) Draw the Bode diagram for this compensator. (5%)  
 (b) Find the maximum phase in terms of  $\alpha$  and the frequency at the maximum phase. (5%)  
 (c) In Fig. 5, if the static velocity error coefficient  $K_v$  and the Bode diagram of open-loop transfer function for the system with  $G_c(s) = 1$  are given, discuss the effects on  $K_v$  and the Bode diagram of open-loop transfer function when the compensator is applied; i.e.,  $G_c(s) = D(s)$ . (5%)

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6. A control system is given as Fig. 6.  
 If define  $x_1 = y$  and  $x_2 = \dot{y}$ , the system can be represented in state-space form as

$$\begin{bmatrix} \dot{x}_1(t) \\ \dot{x}_2(t) \end{bmatrix} = \mathbf{A} \begin{bmatrix} x_1 \\ x_2 \end{bmatrix} + \mathbf{B}r$$

$$y = \mathbf{C} \begin{bmatrix} x_1 \\ x_2 \end{bmatrix}$$

- (a) Find  $\mathbf{A}$ ,  $\mathbf{B}$ , and  $\mathbf{C}$ . (5%)  
 (b) Find the eigenvalues of  $\mathbf{A}$ . (5%)  
 (c) Find the range of  $k$  from the eigenvalues obtained in (b) to guarantee the stability of the system. (5%)  
 (d) Verify your result in (c) by Routh's stability criterion. (5%)

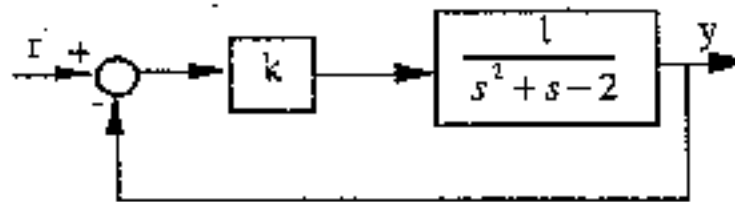


Fig. 6

7. A control system has the structure shown in Fig. 7 where  $u$  and  $y$  are input and output, respectively, and  $x_1$ ,  $x_2$ , and  $x_3$  are states of the system.  
 (a) Test the controllability of the system. (5%)  
 (b) Test the observability of the system. (5%)  
 (c) If the system is controllable, design the state-feedback control law

$$u = [k_1 \quad k_2 \quad k_3] \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} + v$$

so that the closed-loop system has poles at -1, -2, and -3. (5%)

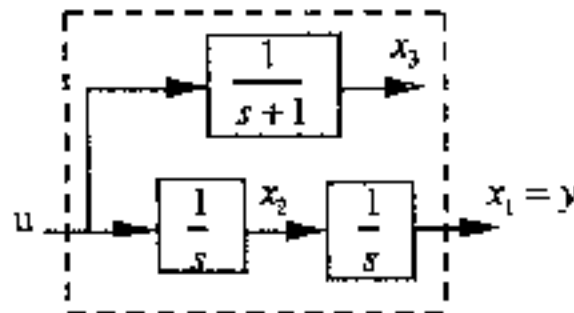


Fig. 7